

Common Unit Conversions

1 kW = 3,412 Btu/h

1 ton = 12,000 Btu/h

1 MMBtu = 1,000,000 Btu = 10 Therms

1 Therm = 100,000 Btu

1 MBH = 1 kBtu/h = 1,000 Btu/h

1 Boiler hp = 33,475 Btu/h

1 psi = 2.31 ft of head = 2.036 in Hg

$P_{Abs} \text{ (psia)} = P_{Gauge} \text{ (psig)} + P_{Atm}$, where $P_{Atm} = 14.7 \text{ psi}$

1 hp = .746 kW = 2,545 Btu/h

*if calculating heat gain from motors, divide by efficiency

$^{\circ}\text{C} = 5/9 * (^{\circ}\text{F} - 32)$ $^{\circ}\text{F} = 9/5 * ^{\circ}\text{C} + 32$ $^{\circ}\text{R} = ^{\circ}\text{F} + 460$

$^{\circ}\text{K} = (^{\circ}\text{F} + 460) * 5/9 = ^{\circ}\text{C} + 273$

Common Formulas

$\text{kW} = \text{Volts} * \text{Amps} * \text{PF} * 1.73 / 1,000$ (for 3 phase power)

$\text{kW of Motor Load} = .746 * \text{hp} * \text{Load Factor} / \text{Motor Efficiency}$

$\text{Airside Sensible Cooling Load (tons)} = 1.08 * \text{cfm} * \Delta T / 12,000$

$\text{Waterside Cooling Load (tons)} = 500 * \text{gpm} * \Delta T / 12,000$

$\text{Airside Heating Load (Btu/h)} = 1.08 * \text{cfm} * \Delta T$

$\text{Waterside Heating Load (Btu/h)} = 500 * \text{gpm} * \Delta T$

$\text{kW/ton} = 12 / \text{EER}$

$\text{EER} = \text{Cooling COP} * 3.412$

$\text{COP} = \text{Work Done} / \text{Energy In}$

$\text{Pump hp} = \text{Ft Head} * \text{gpm} * \text{SG} / 3960 / \text{Pump Eff}$ (SG water = 1)

$\text{Fan hp} = \text{Static Press} * \text{cfm} / 6354 / \text{Fan Eff}$

$\text{Motor kW (when shaft power is known)} = \text{Shaft hp} * .746 / \text{Motor Eff}$

Conductive Heat Transfer (Btu/hr) = $U * A * \Delta T$, where $U = 1/R$ and A is in ft^2

Solar Heat Gain = $\text{SHGF} * A$, where SHGF = Solar Heat Gain Factor

Affinity Laws (P=Power, H=Head/Pressure, Q = flow)

$Q_1/Q_2 = \text{RPM}_1/\text{RPM}_2$ $H_1/H_2 = (\text{RPM}_1/\text{RPM}_2)^2$ $P_1/P_2 = (\text{RPM}_1/\text{RPM}_2)^3$

*Actual exponents are lower than formulas

Common Rules of Thumb

Water Cooled Chiller Efficiency = .5 - .8 kW/ton

Chilled Water Flow = 2.4 gpm/ton

Condenser Water Flow = 3.0 gpm/ton

DX/Air cooled chiller efficiency = 1-1.3 kW/ton

AHU Airflow = 400 cfm/ton

Lighting Power Density = .75 to 1.5 watts/sf (office and similar)

Equivalent Full Load Hours (EFLH) = 350 – 600 hrs for non-process cooling in PNW.

CW Reset = A decrease in condenser water temp will save between .75% and 1.5% per deg at the chiller. Constant speed chillers are closer to .75%, VFD closer to 1.5%. The same applies to refrigerant temperature in a refrigeration system.

CHW Reset (Const Speed Chiller) = An increase in chilled water temperature will save approximately .75% per deg at the chiller.

CHW Reset (VFD Chiller) = An increase in chilled water temperature will save approximately 2% per deg at the chiller. Note that if the coils have two way valves, this could increase pump power.

Suction Temperature Reset = An increase in refrigerant suction temperature will save approximately 2% in compressor power per degree.

EUI = A typical existing commercial building in the PNW may be in the range of 75-125 kBtu/sqft. New construction will be closer to 50-60 kBtu/sqft.